

# Direct Numerical Simulations of Wave-Current Boundary Layers Over Bumpy walls

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# Motivation

Wind-Waves



$$Re_w = \frac{U_b^2}{\omega \nu}$$

Wave Reynolds  
number

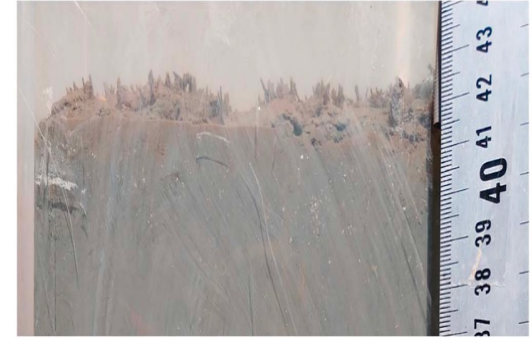
Tidal Currents



$$Re_* = \frac{u_* H}{\nu}$$

Flow Reynolds  
number

“Bumpy” bottom



$$\bar{k}_s = f(\psi_r(x_3))$$

Mean roughness  
height

$$\overline{\tau(t)} = f(Re_w, Re_*, \bar{k}_s, \dots)$$

## Goal

$$C_d = f(Re_w, Re_*, \bar{k}_s, \dots)$$

SUNTANS, SWASH, DELFT-3D, etc.

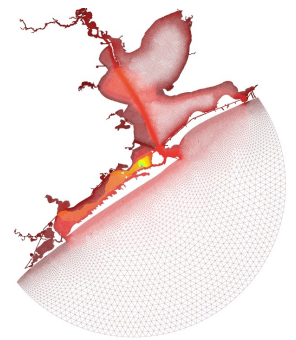


Photo Source

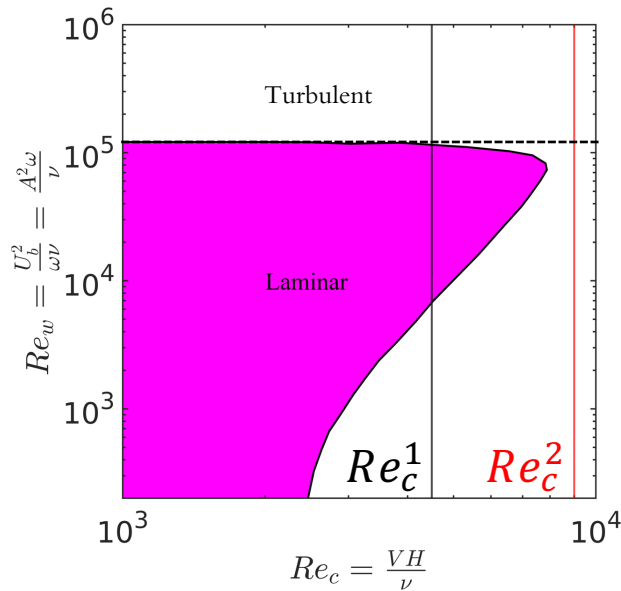
1. <https://sites.google.com/site/thebrockeninglory/home?authuser=0>
2. <https://i.ytimg.com/vi/FfK4HvRcFy/maxresdefault.jpg>
3. Egan et al. (2019)

Photo Source:

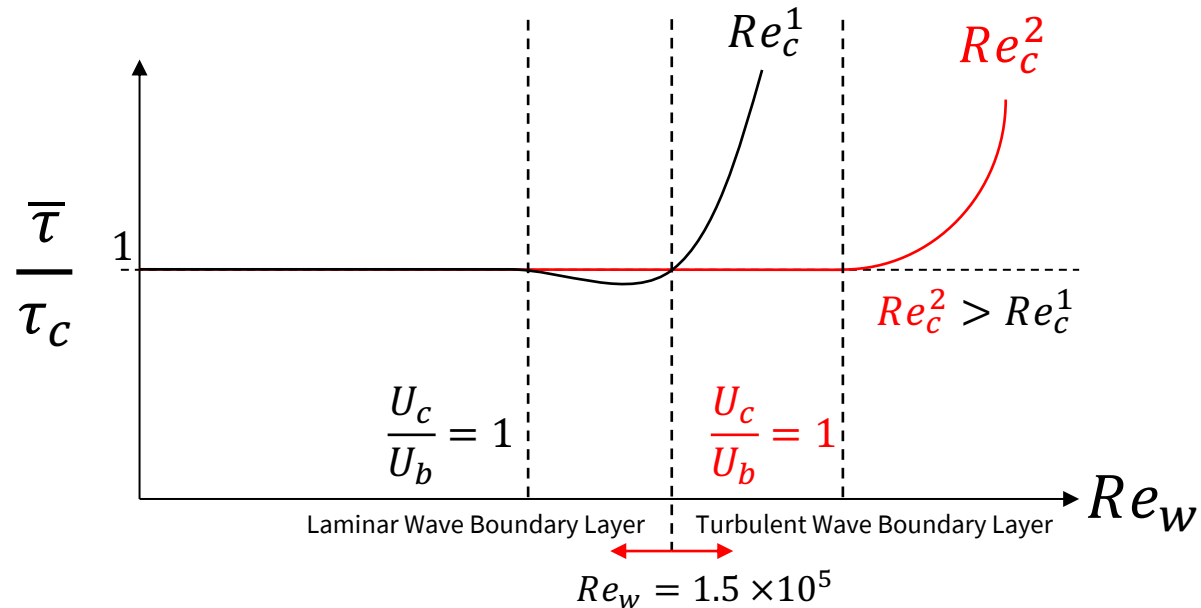
<https://i.ytimg.com/vi/FfK4HvRcFy/maxresdefault.jpg>

# What do we know already?

Wave-Current boundary layer over *flat walls* well understood (Lodahl et al. 1998; Scotti & Piomelli 2001; Manna et al. 2012, 2015; Nelson & Fringer 2018)



Source: Lodahl et al. (1998)



## What about “bumpy” walls?

- Most studies are experimental
- Grant & Madsen (1979) analytical theory
- Only exception is the work by Bhaganagar (2008) – 1<sup>st</sup> order statistics

# Problem Setup

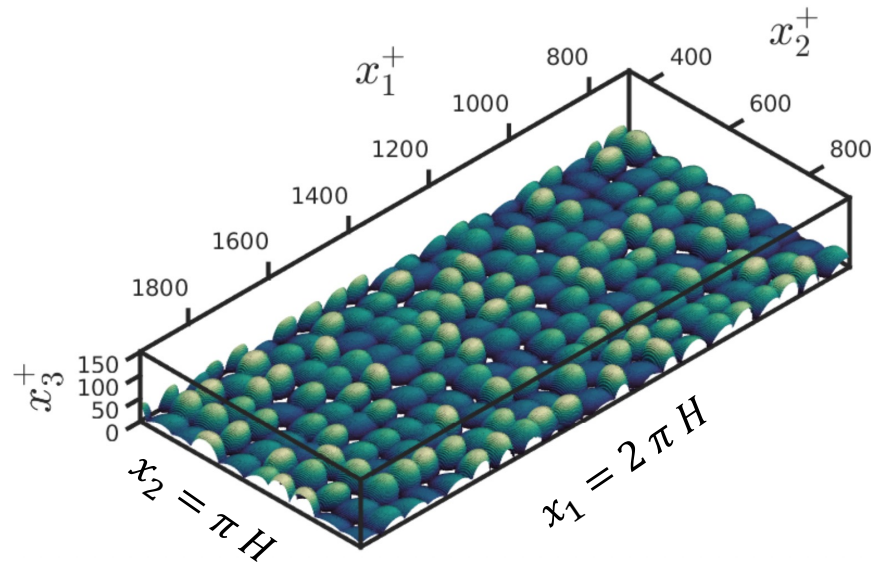
Staggered, second-order accurate, finite-difference method (Orlandi 2000, Moin & Verzicco 2016)  
 IBM implemented over the fluid solver developed by Lozano-Duran & Bae (2016,2019)

$$\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} = -\frac{1}{\rho_o} \frac{\partial p}{\partial x_i} + \nu \frac{\partial^2 u_i}{\partial x_j \partial x_j} + \underbrace{U_b \omega \cos(\omega t)}_{\text{Waves}} \delta_{i,1} + \underbrace{\Pi_c}_{\text{Current}} \delta_{i,1} + \underbrace{F_{IBM}}_{\text{Roughness}} \quad \& \quad \frac{\partial u_i}{\partial x_i} = 0$$

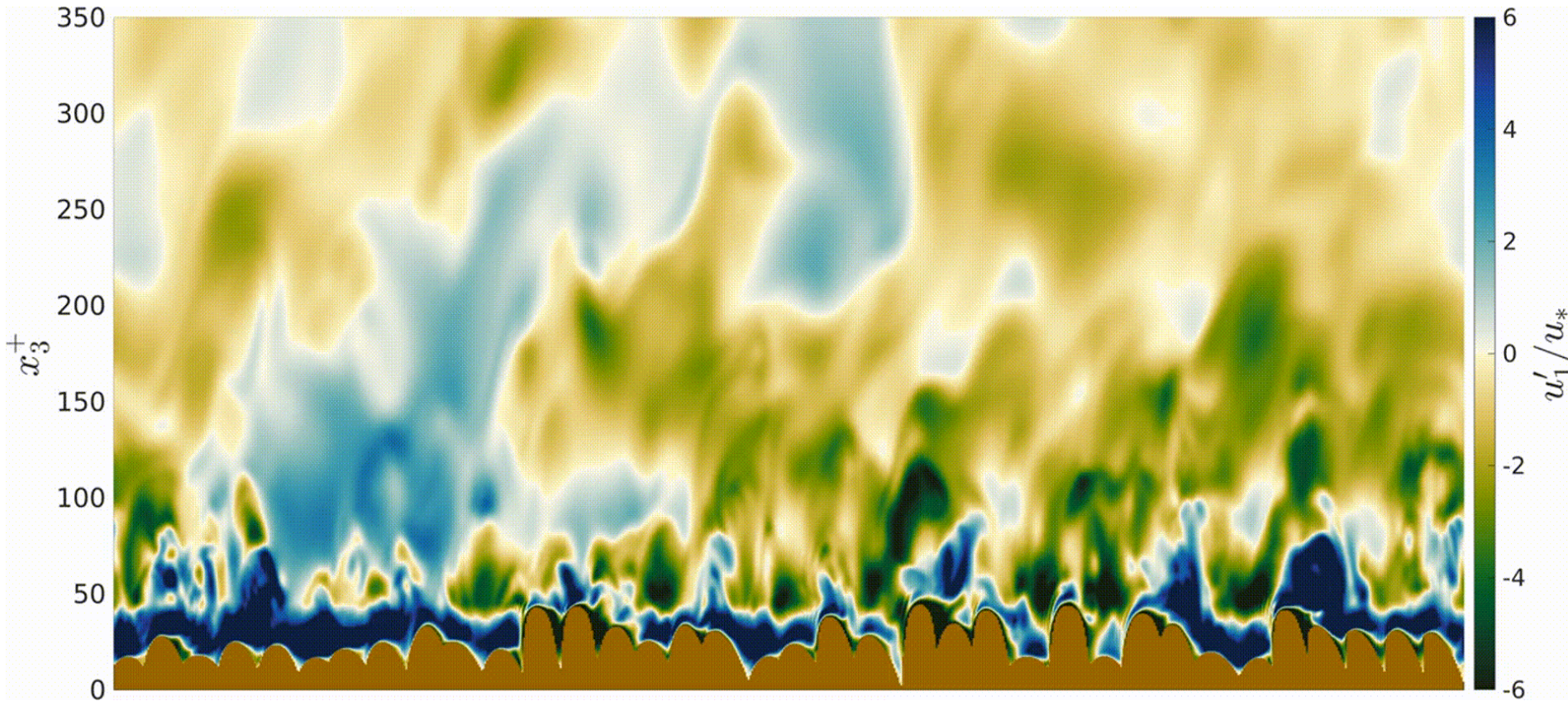
$$u_3(x_3 = H) = 0, \& \frac{\partial u_i}{\partial x_3}(x_3 = H) = 0 \quad \forall i \in \{1,2\}$$

Non-dimensional  
 wall coordinates

$$x_3^+ = \frac{x_3 u_*}{\nu}$$



Channel depth is 0.1 m = 10 cm



# NAMING CONVENTION

*W* – Waves

*C* – Currents

*350* – Friction Reynolds Number

*F* – Flat Walls

*B* – Bumpy Walls

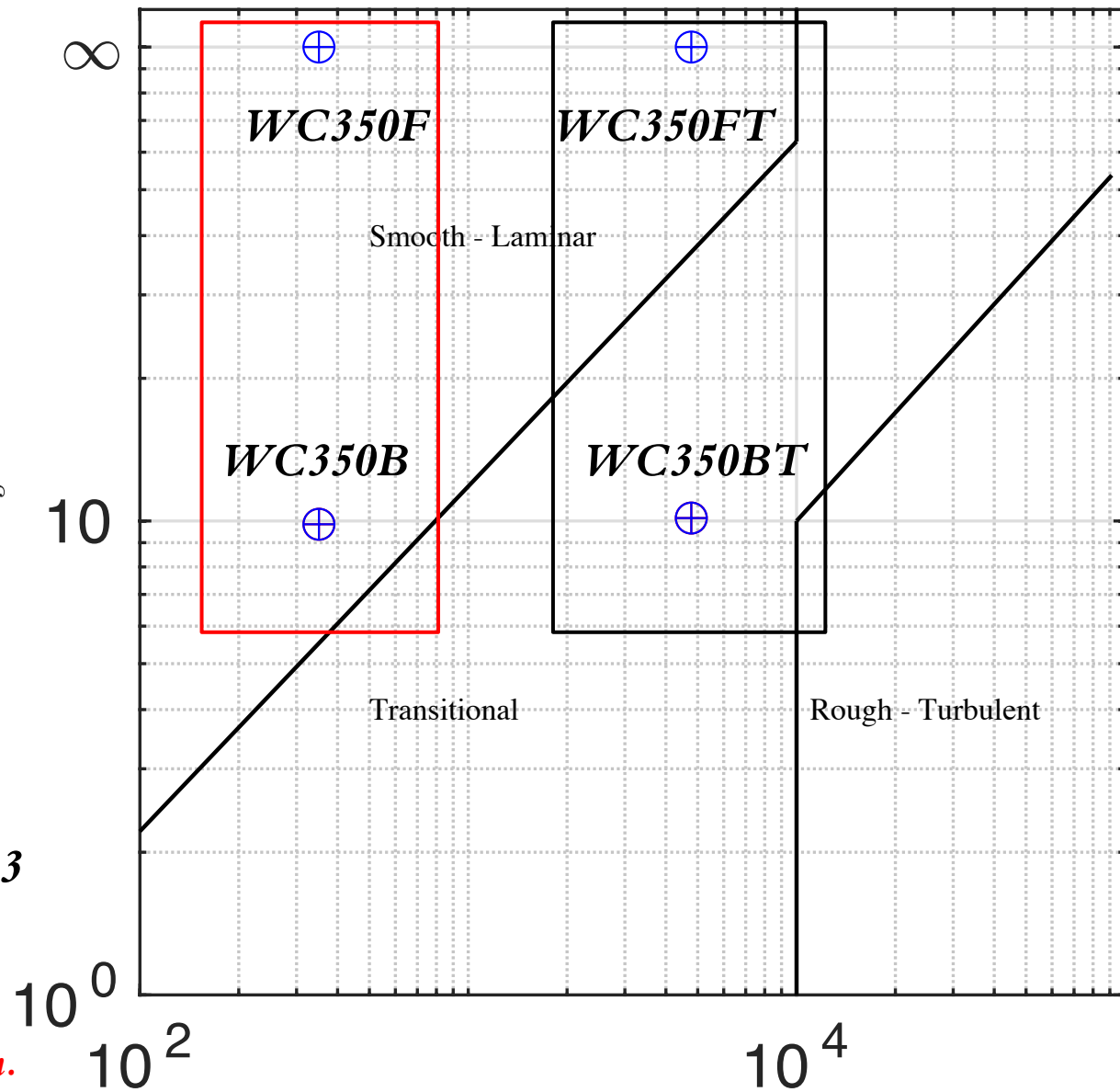
*T* – Transitional wave conditions

$$A = \frac{U_b}{\omega} \quad A \parallel k_s$$

- WC350F & WC350B – 2 days for 200 wave periods using 64 MPI tasks
- WC350FT & WC350BT – 3 months using 256 MPI tasks

• *Patil, A., Fringer, O., (Submitted Jan. 2022), Journal of Fluid Mechanics*

• *Patil, A., Monismith, S., Fringer, O., (In Prep)*



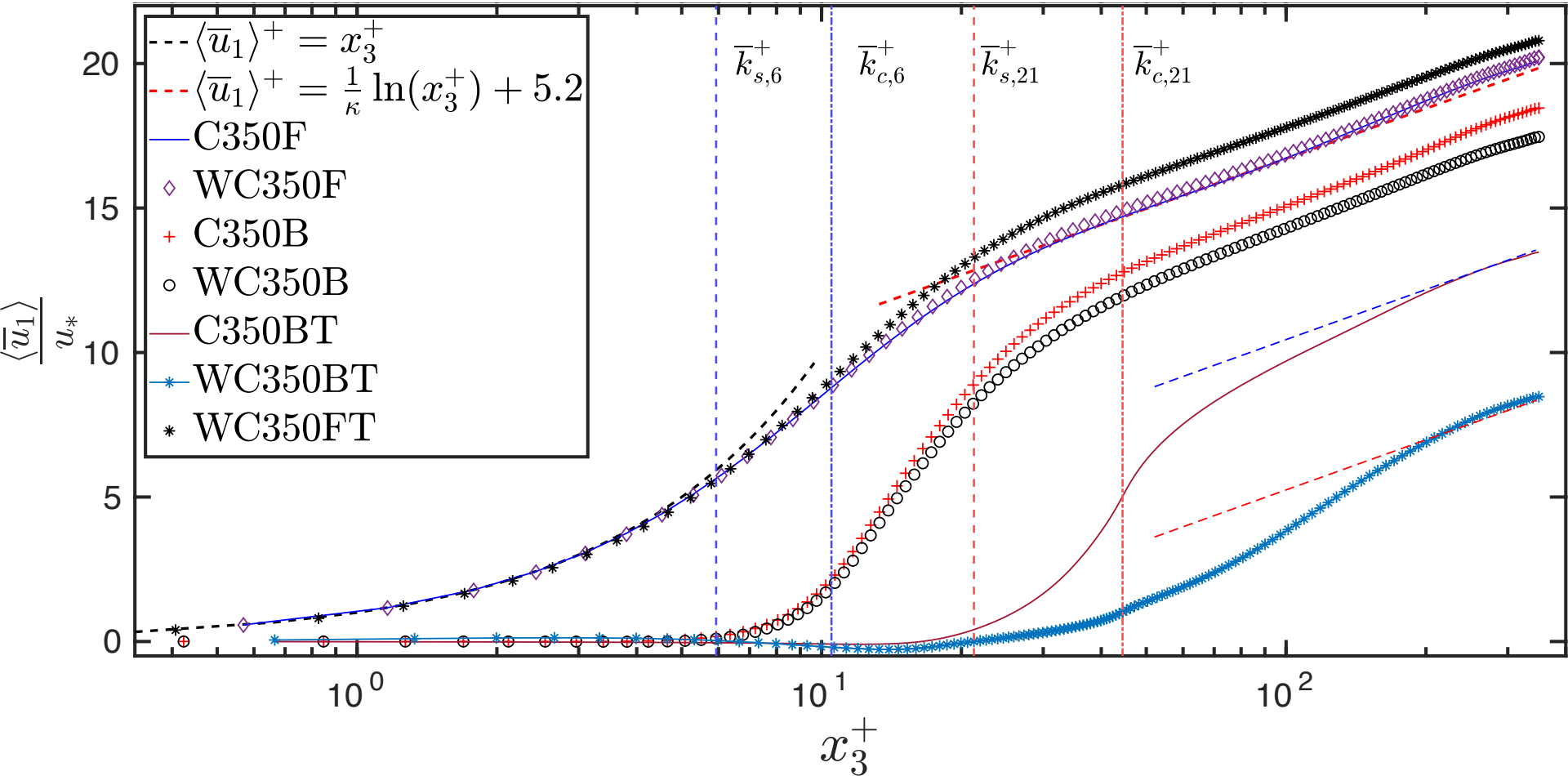
$$Re_w = \frac{U_b^2}{\omega \nu} = \frac{A^2 \omega}{\nu}$$

# Time- and Planform-averaged Velocity Profile

Log-law (Raupach et al. 1991)

$$\frac{\langle \bar{u}_1 \rangle}{u_*} = \frac{1}{\kappa} \ln \left( \frac{x_3 - \bar{k}_s}{z_0} \right)$$

$$z_0 = \begin{cases} \frac{\bar{k}_s}{\nu} & \dots \bar{k}_s > 0 \\ \alpha_k \nu & \dots \bar{k}_s = 0 \\ \frac{9u_*}{\nu} & \dots \bar{k}_s = 0 \end{cases}$$



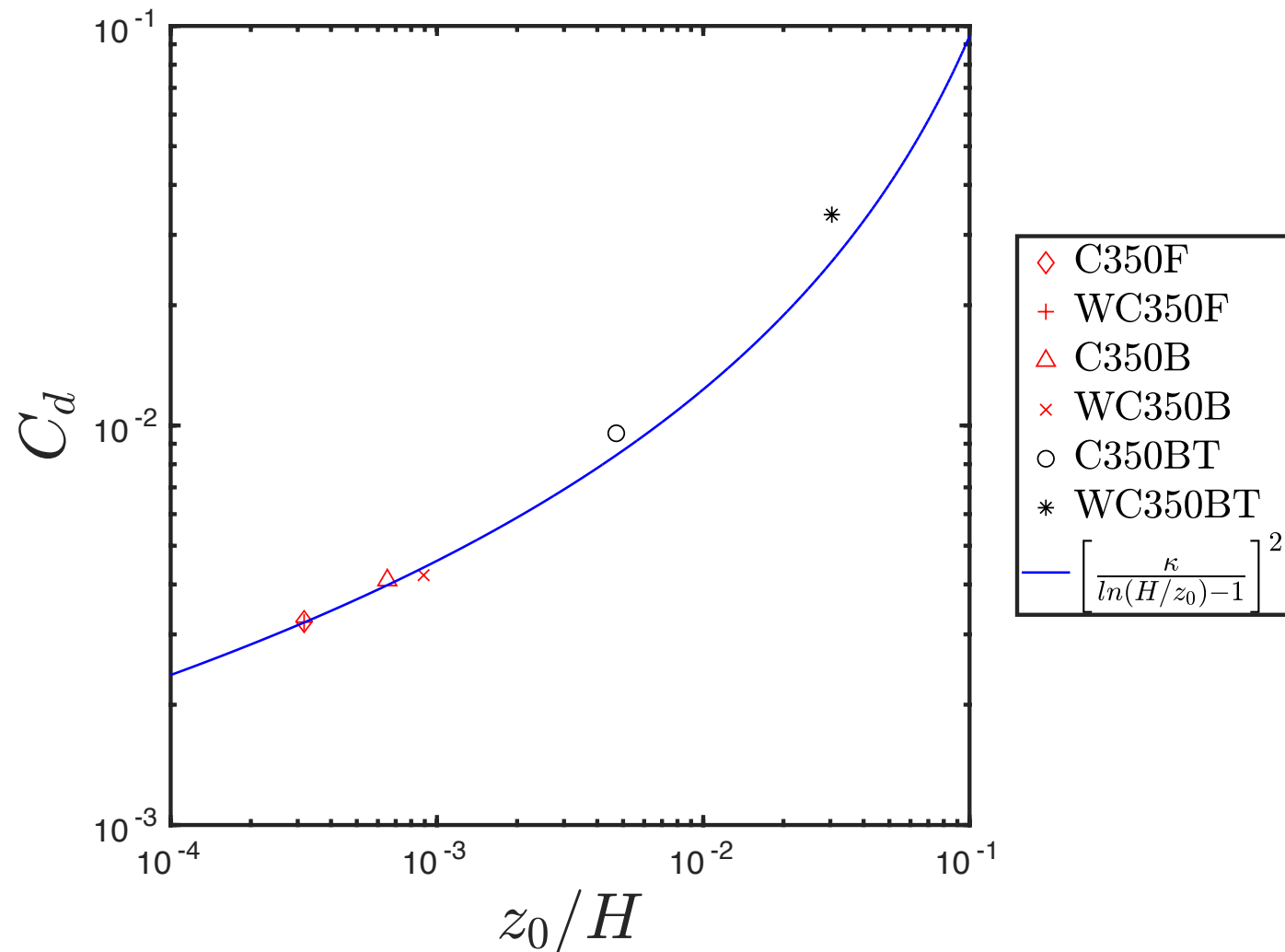
# Drag Coefficient

Exact Definition

$$C_d^* = \frac{u_*^2}{\langle \bar{u}_1 \rangle_v^2}$$

Assuming log-law holds

$$C_d = \left[ \frac{\kappa}{\ln\left(\frac{H}{z_0}\right) - 1} \right]^2$$

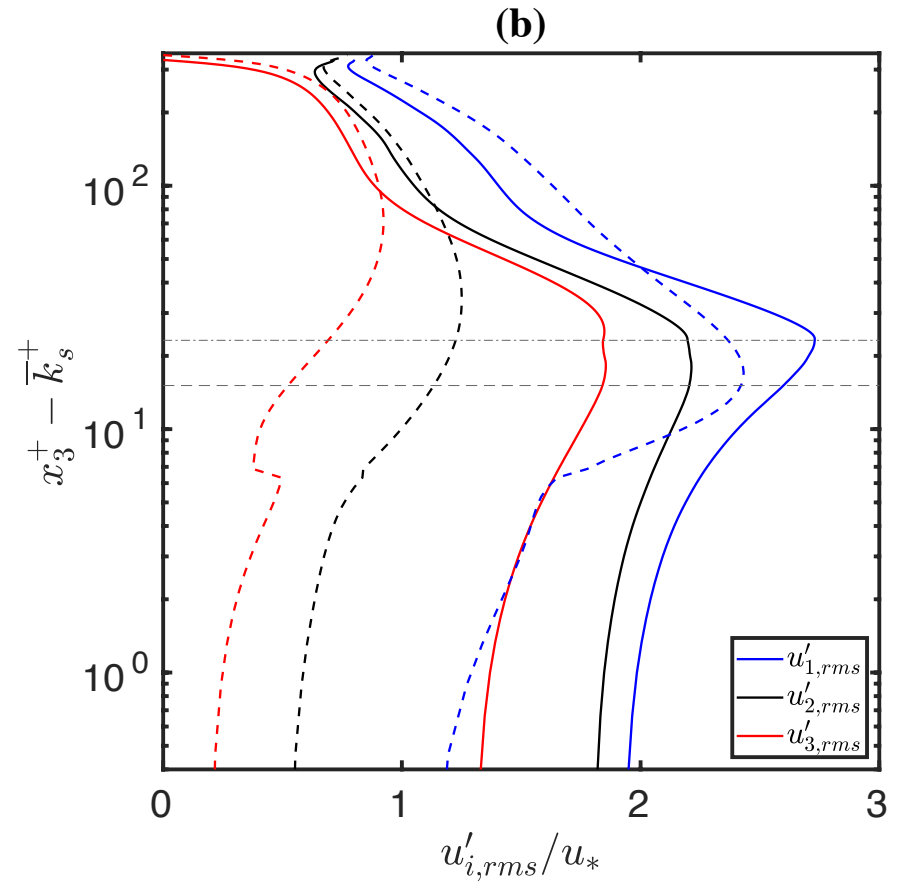
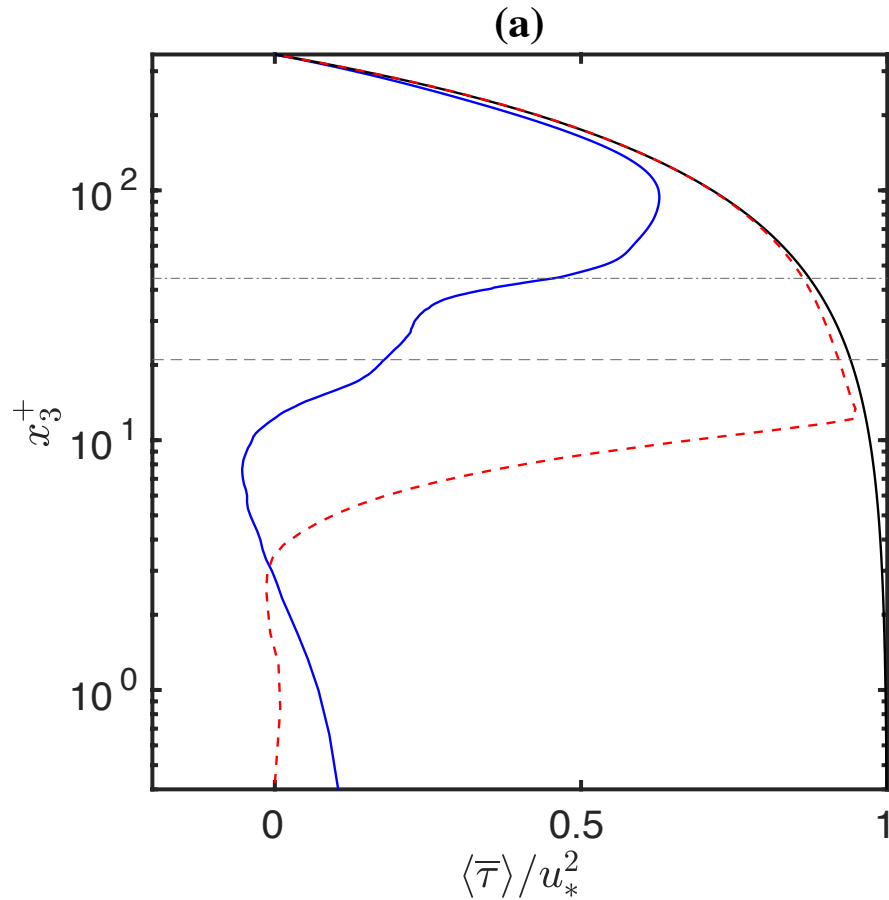




# Time-averaged Stress & RMS velocity

—	WC350BT
- - -	WC350B

$$0 = \Pi_c - \frac{\partial \langle \bar{\tau} \rangle}{\partial x_3}$$



# Key Points

- Wave-Current BL's (Flat walls) – No change – Current dominated, hydraulically smooth ( $\frac{\bar{k}_s}{\delta_w} < 4$ )
- Wave-Current BL's (Bumpy walls) – Drag coefficient increases (3\*% to 11%) – Current dominated, hydraulically smooth
- One-way coupled flow (Scotti & Piomelli 2001; Manna et al. 2012, 2015; Nelson & Fringer 2018)
- ***Grant & Madsen (1979) theory holds despite the various assumptions that demonstrably do not hold***
- Enhanced dissipation with simultaneous decreased mean shear production of TKE
- Pressure-Strain rate correlations important

*THANK YOU!*



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**XSEDE**

XSEDE Project: CTS190063

